

TECHNICAL NOTES 5

CRUSHERS

5.1 Jaw and Gyratory Crushers.

Jaw and gyratory crushers are used mostly for primary crushing. They are characterized by wide gape and narrow discharge and are designed to handle large quantities of material.

The capacity of the crusher is determined by its size.

The gape determines the maximum size of material that can be accepted. Maximum size that can be accepted into the crusher is approximately 80% of the gape.

Jaw crushers are operated to produce a size reduction ratio between 4:1 and 9:1. Gyratory crushers can produce size reduction ratios over a somewhat larger range of 3:1 to 10:1.

The primary operating variable available on a crusher is the set and on jaw and gyratory the open-side set (OSS) is specified. This reflects the fact that considerable portions of the processed material fall through the crusher at OSS and this determines the characteristics size of the product. The set of a crusher can be varied in the field and some crushers are equipped with automatically controlled actuated for the automatic control of the set. The open- and closed-side sets and the gape are identified in Figure 5.1. The throw of the crusher is the distance that moving jaw moves in going from OSS to CSS.

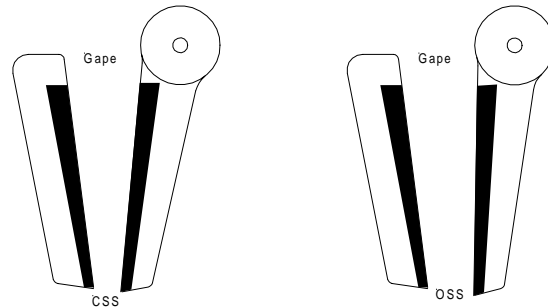


Figure 5.1 Schematic diagram of a crusher showing the open- and closed-side settings.

$$\text{Throw} = \text{OSS} - \text{CSS}.$$

The capacity is a function of size and OSS. Manufacturers publish tables of capacity for their crushers of various size as a function of the open-side set.

5.1.1 Cone crushers

Cone crushers are commonly used for secondary, tertiary and quaternary crushing duties. Two variations are available - standard and short head

The chief difference between cone and gyratory or jaw crushers is the nearly parallel arrangement of the mantle and the cone at the discharge end in the cone crusher. This is illustrated in Figure 5.2.

Reduction ratios in the following ranges are common for cone crushers:

6:1 - 8:1 for secondaries

4:1 - 6:1 for tertiary and quaternary crushing.

The size distribution of the products tends to be determined primarily by the CSS since no particle can fall through during a single open side period and all particles will experience at least one closed side nip.

The CSS is adjusted by screwing the bowl up or down.

5.1.2 Impact crushers

Breakage is achieved by impact using either hammer action on the individual particles or by sudden impact from a high velocity trajectory.

High reduction ratios of between 20:1 and 40:1 can be achieved with hammer type impact crushers.

Only low reduction ratios of about 2:1 can be achieved with kinetic energy type impact crushers.

5.1.3 Crushing mechanisms and product size distributions.

The crushing action of a crushing machine is described most usefully through the classification - breakage cycle model. The operation of a crusher is periodic with each period consisting of a nipping action and an opening action. During the opening part of the cycle material moves downward into the crusher and some material falls through and out. A certain amount of fresh feed is also taken in. This is illustrated in Figure 5.3.

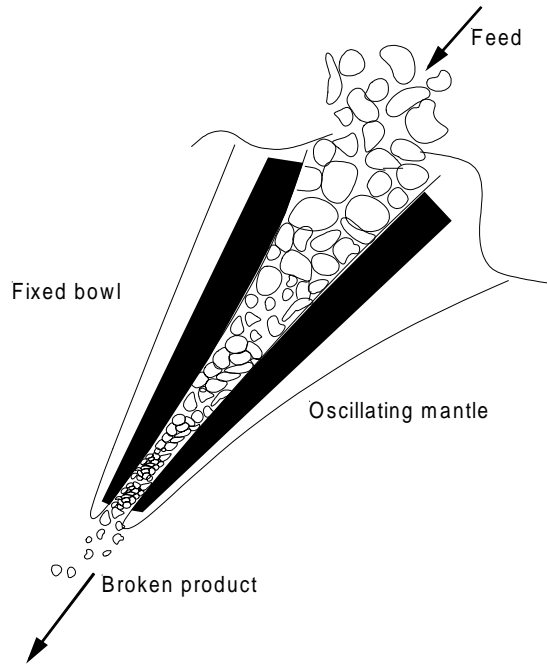


Figure 5.2 Schematic view of the crushing zone of the cone crusher.

Let us now describe this behavior quantitatively. It is best to work with a discrete size distribution. So define

- p_i^F = fraction of the feed in size class i ,
- p_i = fraction of the product in size class i ,
- M = mass of material held in the crusher,
- b_{ij} = fraction of particles breaking in size class j by that end up in size class i .
- m_i = fraction of material in the crusher in size class i
- $c_i = c(d_i)$

= fraction of material in size class i that is retained for breakage during the next nip of

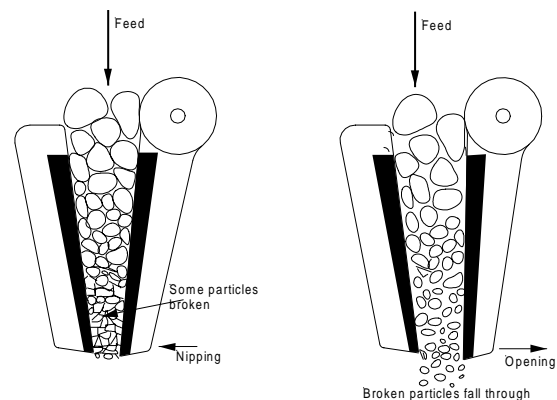


Figure 5.3 The opening and nipping cycles in the crusher on which the model is based.

the crusher.

W = Mass of total feed that is accepted during a single opening
 = mass of product discharged.

Size class 1 contains the largest particles.

Let us follow the fortunes of material in the largest size class starting with an amount Mm_1 in the crusher.

During an opening phase of the cycle:

Material discharged from the crusher = $(1-c_1)Mm_1$

Material positioned for breakage in the breakage zone during next nip = c_1Mm_1

Accepted from feed = Wp_1^F

After the next nip the crusher must again have an amount Mm_1 in the crushing zone since the operation is at steady state:

$$Mm_1 = Wp_1^F + c_1Mm_1b_{11}$$

$$\frac{Mm_1}{W} = \frac{p_1^F}{1 - c_1b_{11}} \quad (5.1)$$

Product discharged = $Wp_1 = (1-c_1)Mm_1$

$$p_1 = (1-c_1)\frac{Mm_1}{W} \quad (5.2)$$

Now consider the next size down:

During an opening phase of the cycle:

Material discharged = $(1-c_2)Mm_2$

Material positioned for breakage during next nip = c_2Mm_2

Accepted from feed = Wp_2^F

After next nip:

$$Mm_2 = Wp_2^F + c_2Mm_2b_{22} + c_1Mm_1b_{21}$$

$$\frac{Mm_2}{W} = \frac{1}{1 - c_2b_{22}} \left(p_2^F + c_1\frac{Mm_1}{W}b_{21} \right) \quad (5.3)$$

Product discharged = $Wp_2 = (1-c_2)Mm_2$

$$p_2 = (1-c_2)\frac{Mm_2}{W} \quad (5.4)$$

The next size down can be handled in exactly the same way to give

$$Mm_3 = Wp_3^F + c_3Mm_3b_{33} + c_2Mm_2b_{32} + c_1Mm_1b_{31}$$

$$\frac{Mm_3}{W} = \frac{1}{1-c_3b_{33}} \left(p_3^F + c_2 \frac{Mm_2}{W} b_{32} + c_1 \frac{Mm_1}{W} b_{31} \right) \quad (5.5)$$

This procedure can be continued from size to size.

In general

$$\frac{Mm_i}{W} = \frac{1}{1-c_i b_{ii}} \left(p_i^F + \sum_{j=1}^{i-1} c_j \frac{Mm_j}{W} b_{ij} \right) \quad (5.6)$$

The series of equations (5.6) can be easily solved recursively for the group Mm_i/W starting from size class number 1. The size distribution in the product can then be calculated from

$$p_i = \frac{(1-c_i)Mm_i}{W} = \frac{1-c_i}{1-c_i b_{ii}} \left(p_i^F + \sum_{j=1}^{i-1} c_j \frac{Mm_j}{W} b_{ij} \right) \quad (5.7)$$

And the distribution of sizes in the product is completely determined from the size distribution in the feed and a knowledge of the classification and breakage functions.

The classification function is usually of the form shown in Figure 5.4.

d_1 and d_2 are parameters that are characteristic of the crusher. They are determined primarily by the setting of the crusher. Data from operating crusher machines indicate that both d_1 and d_2 are proportional to the closed side setting. d_1 is the smallest size particle that can be retained in the crushing zone during the opening phase of the cycle. d_2 is the largest particle that can fall through the crushing zone during the opening phase of the cycle.

A useful form of the classification function is

$$c_i = 1 - \left(\frac{d_{pi} - d_2}{d_1 - d_2} \right)^n \quad \text{for } d_1 < d_{pi} < d_2$$

$$= 0 \quad \text{for } d_{pi} \leq d_1$$

$$= 1 \quad \text{for } d_{pi} \geq d_2 \quad (5.8)$$

For both standard and short-head Symons cone crushers,

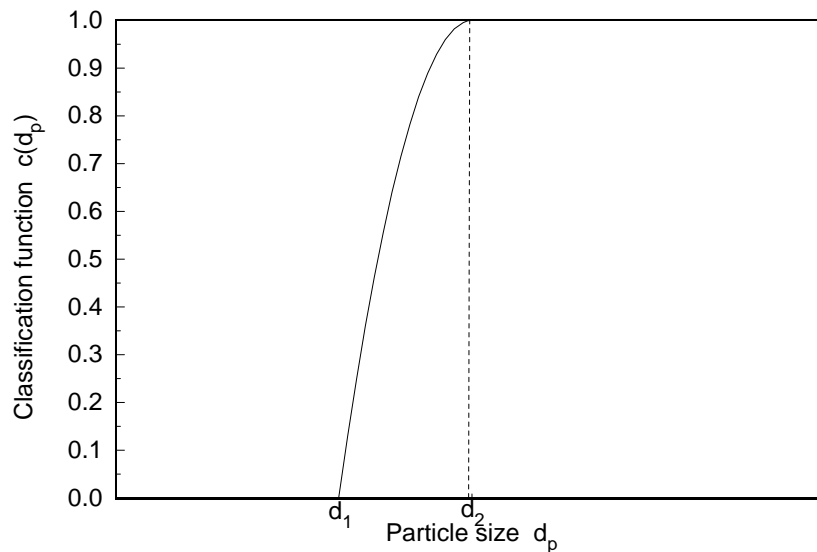


Figure 5.4 A typical internal classification function for a crusher

$$d_1 = \alpha_1 CSS \quad (5.9)$$

$$d_2 = \alpha_2 CSS + d^* \quad (5.10)$$

α_1 varies from about 0.5 to 0.95 and α_2 varies from about 1.7 to 3.5. n is usually approximately 2 but can be as low as 1 and as high as 3. Higher values of n usually require higher values of α_2 . d^* is usually set to 0.

Breakage functions of the type

$$B(x;y) = K \left(\frac{x}{y} \right)^{n_1} + (1-K) \left(\frac{x}{y} \right)^{n_2} \quad (5.11)$$

are normally used to describe crusher behavior.

The values of b_{ij} can be obtained from the cumulative breakage function by

$$b_{ij} = B(D_{i-1}; d_{pj}) - B(D_i; d_{pj}) \quad (5.12)$$

and

$$b_{jj} = 1 - B(D_j; d_{pj}) \quad (5.13)$$

represents the fraction of material that remains in

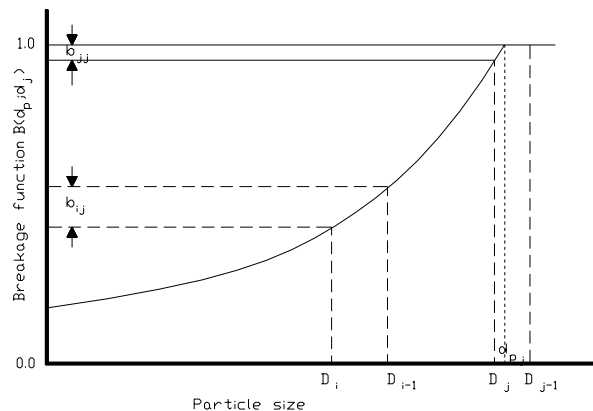


Figure 5.5 The breakage function for crushing machines. This function has a value 1 at the representative size of the parent class. Compare this with the breakage function used for grinding machines.

size interval j after breakage. These relationships are illustrated in figure 5.5. n_1 is approximately 0.5 for both standard and short-head crushers and n_2 is approximately 2.5 for short-head and 4.5 for standard crushers.

The parameters in the classification and breakage functions are obviously specific to the type and size of crusher. Unfortunately not many studies have been done to establish their values under a range of actual operating conditions. In practice it is often necessary to estimate them from measured particle distributions in the products from operating crushers. Once established for a particular material in a particular crusher, they should be independent of the closed side set. This allows the crusher performance to be simulated at the various CSS.

Model based on:

1. Whiten W.J. Walter G.W. and White M.E. A breakage function suitable for crusher models. 4th Tewkesbury Symposium, Melbourne (1979) p 19.1 - 19.3.
2. Whiten W.J. The simulation of crushing plants. Application of computer methods in the mineral industry. Apcom 10 S.Afr. Inst of Mining and Metall. Johannesburg (1973) p 317-323.
3. Karra V.K. A process performance model for ore crushers. Proc. 4th Int. Min. Proc. Congress Toronto (1982) III p 6.1-6.14.

Table 3.1 Approximate capacities of jaw crushers in tonnes/hr.

The size designation used here is the traditional one in which the feed opening is specified as gape \times length in inches.

Size Feed opening	Max rpm of flywheel	Motor kW	Open-side setting mm								
			25	32	38	51	63	76	102	127	152
10×20	300	15	12.7	15.4	18.2	23	31				
10×24	275	11	14.5	17.3	20	23	30				
15×24	275	22		20.9	24.5	31	38.1	45.4			
14×24	275	19			23.6	30	37.2	45.4			
24×36	250	56				70	86.3	103	136		
30×42	200	75					113	136	118 2	227	272
			Open side setting mm								
			63	76	102	127	152	178	203	229	254
32×42	200	75			227	263	300	327	363		
36×48	180	93		189	245	300	354	409			
42×48	180	110				345	381	426	463	490	527

48×60	170	150					436	481	517	554	600
56×72	120	186						454	500	567	617
66×84	90	225						700	772	863	950

Table 3.2 Approximate capacities of gyratory crushers in tonnes/hr.

Size is specified as gape × lower mantle diameter in inches.

Size	Speed rpm	Motor kW	Open-side setting mm									
			51	63	76	89	102	114	127	140	152	
30×60	425	150	313	381	450	508	567	630	695	760		
30×55	600	300		381	463	518	590	663	735	817		
36×60	375	186		458	540	604	680	755	830	900	970	
42×65	514	400						800	908	1017	1317	1500
			Open-side setting mm									
			127	140	152	178	190	203	216	229	241	254
42×70	380	300	708	790	863	944	1017	1090				
48×74	514	500	1544	1680	1816	1952	2088	2452				
48×80	330	500			1376	1462	1562	1662	1770	1870		
54×74	514	500		1634	1771	1907	2043	2180	2315			
54×80	330	500			1307	1394	1490	1580	1680	1770		
60×89	514	600			2270	2424	2580	2815	2960	3270		
			Open-side setting mm									
			190	203	216	229	241	254	267	279	292	305
60×102	300	800		2542	2760	2974		3396		3827		4254
60×109					3904	4195	4485	4776	5067	5357	5675	5993

Table 3.3 Approximate capacities of standard Symons cone crushers in tonnes/hr.

Open-circuit operation.

Size (Max. power kW)	Type of cavity	Feed opening on the closed side* with minimum CSS mm	Closed-side setting mm										
			6	9	13	16	19	22	25	31	38	51	64
2 ft (22)	Fine	57	16	18	23	27	32	36	41	45	54		
	Coarse	83		18	23	27	32	41	45	54	68		
	Extra coarse	100			23	27	36	45	50	63	72		
3 ft (56)	Fine	83		45	59	72	81	91					
	Coarse	119			59	72	91	99	118	136	163		
	Extra coarse	163					99	109	118	136	163		
4 ft (93)	Fine	127		63	91	109	127	140	154	168			
	Medium	156			99	118	136	145	163	181	199		
	Coarse	178					140	154	181	199	245	308	
	Extra coarse	231							190	208	254	317	
4½ ft (112)	Fine	109			109	127	145	154	163	181			
	Medium	188				131	158	172	199	227	264		
	Coarse	216					172	195	217	249	295	349	
	Extra coarse	238							236	272	303	358	
5½ ft (150)	Fine	188				181	204	229	258	295	326		
	Medium	213						258	290	335	381	417	
	Coarse	241							290	354	417	453	635
	Extra coarse	331									431	476	680
7 ft (224) (260 EHD)	Fine	253					381	408	499	617	726		
	Medium	303							607	726	807	998	
	Coarse	334								789	843	1088	1270
	Extra coarse	425									880	1179	1380
10 ft (450)	Fine	317						934	1179	1469	1632		
	Medium	394								1570	1814	2267	
	Coarse	470									1905	2449	2857
	Extra coarse	622									1995	2630	3084

EHD = extra heavy duty.

Table 5.4 Approximate capacities of short head Symons cone crushers in tonnes/hr.

Open-circuit operation.

Size (Max power kW)	Type of cavity	Recommended minimum CSS mm	Feed opening with minimum CSS mm		Closed-side setting mm													
			Closed side	Open side	3	5	6	10	13	16	19	25						
2 ft (22)	Fine	3	19	35	9	6	18	27	36									
	Coarse	5	38	51									16	22	29	41		
3 ft (56)	Fine	3	13	41	27	41	54	68	91	99								
	Medium	3	33	60									27	41	68	91	99	
	Extra coarse	5	51	76									59	72	95	113	127	
4 ft (93)	Fine	5	29	57		50	77	86	122	131								
	Medium	8	44	73									91	131	145			
	Coarse	13	56	89									140	163	181			
	Extra coarse	16	89	117									145	168	190	217		
4½ ft (112)	Fine	5	29	64		59	81	104	136	163								
	Medium	6	54	89									104	136	163			
	Coarse	8	70	105									109	158	181	199	227	
	Extra coarse	13	98	133									172	190	254	238		
5½ ft (150)	Fine	5	35	70		91	136	163	208	254	281							
	Medium	6	54	89									163	208	254	281	308	598
	Coarse	10	98	133									190	254	281	308	598	653
	Extra coarse	13	117	133									254	281	308	598	653	
7 ft (224) (260 EHD)	Fine	5	51	105		190	273	326	363	408								
	Medium	10	95	133									354	408	453	506		
	Coarse	13	127	178									453	481	544	598		
	Extra coarse	16	152	203									506	589	653			
10 ft (450)	Fine		76	127			635	735	816	916	1106							
	Medium		102	152									798	916	1020	1224		
	Coarse		178	229									1125	1324	1360			
	Extra coarse		203	254									1478					

EHD = extra heavy duty.